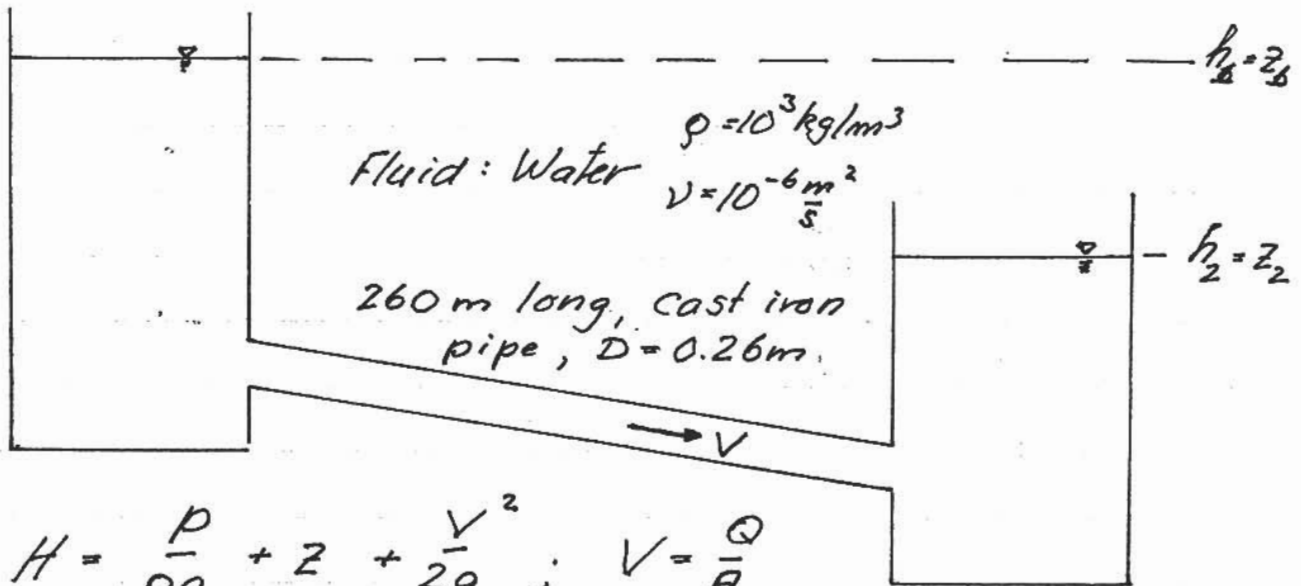


LECTURE #17

1.060 ENGINEERING MECHANICS II

ANALYSIS OF PIPE FLOW

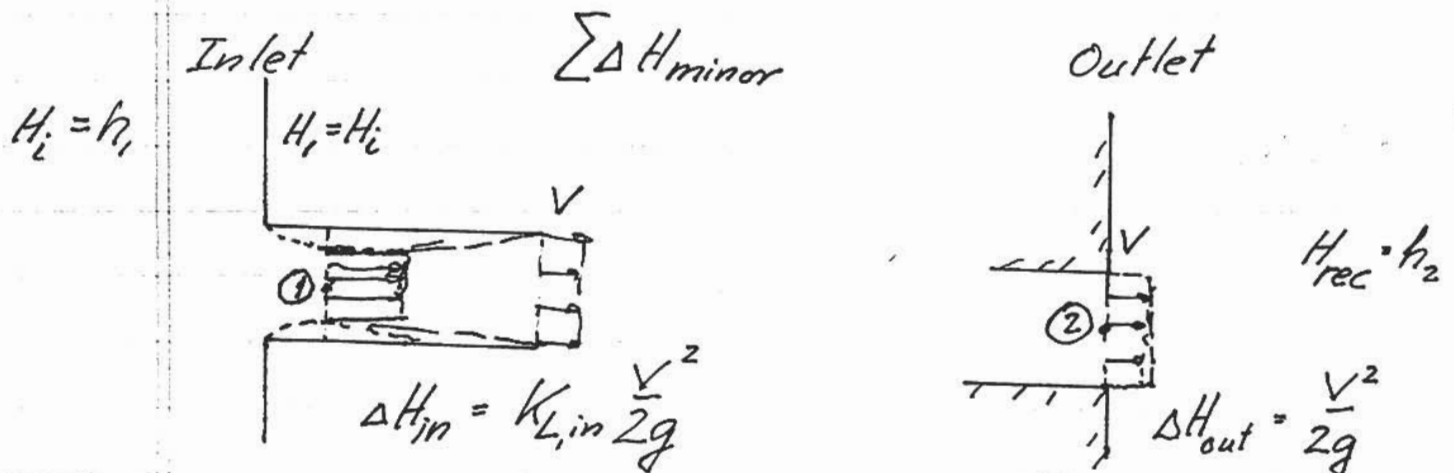


Head in supply tank =  $H_i = h_1$   
 Head in receiving tank =  $H_{rec} = h_2$

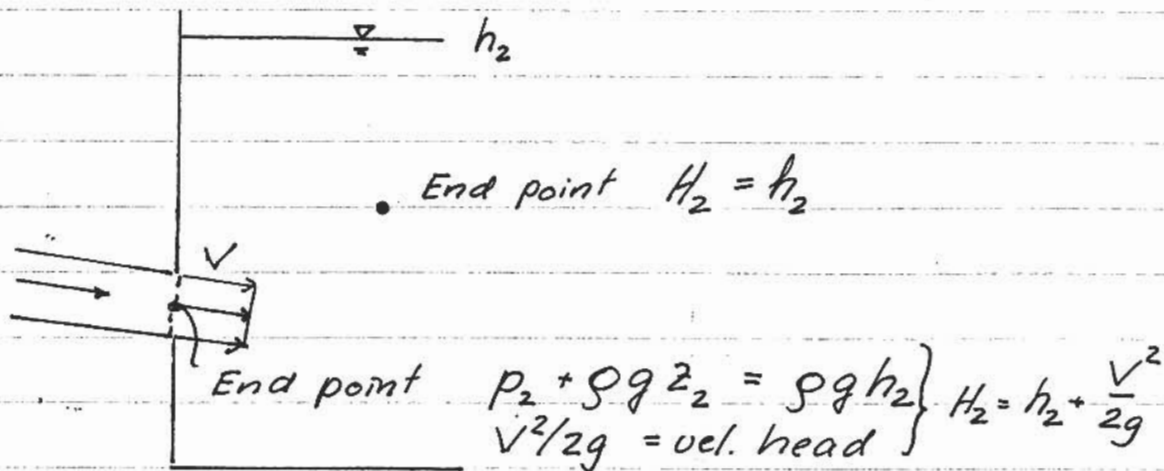
Total headloss in system =  $\Delta H = h_1 - h_2$

$$\Delta H = \sum \Delta H_f + \sum \Delta H_{\text{minor}}$$

Here :  $\Delta H_f = f \frac{L}{D} \frac{v^2}{2g}$  (only one pipe)

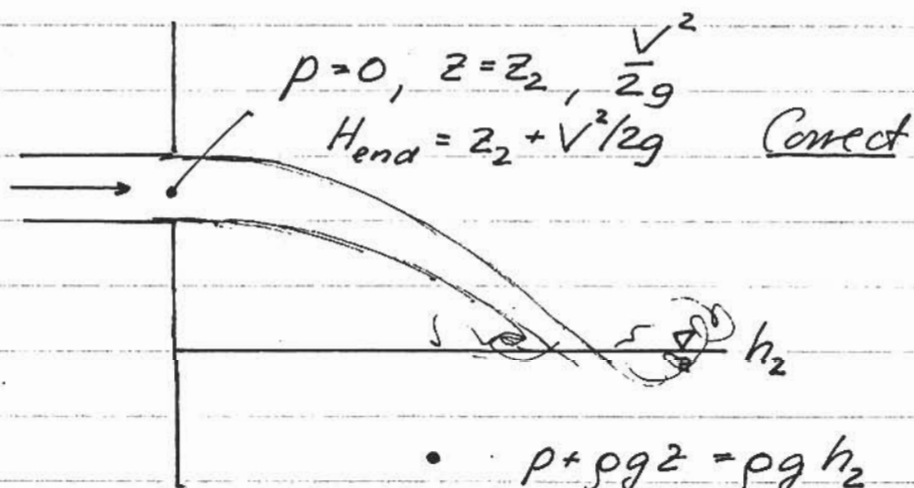


## Choice "End" point



Difference is (of course) the velocity head  $\frac{v^2}{2g}$  which is the head loss of the outflow into a large reservoir!

But if outflow is free then what?



$p + \rho g z = \rho g h_2$   
 $H_{\text{end}} = h_2$  Incorrect

Difference =  $(z_2 - h_2) + \frac{v^2}{2g}$  is NOT just the velocity head! Reason: Impact of free jet into pool causes dissipation.

Proper Choice: Take "end" at exit from pipe

If  $H_{\text{end}} = h_2$  lowering pool level would increase  $h_1, h_2$  and increase flow, which is ABSURD!

$$H_i = H_i = h_i = H_{rec} + \Delta H = h_2 + K_{L,in} \frac{V^2}{2g} + K_{L,out} \frac{V^2}{2g} + f \frac{L}{D} \frac{V^2}{2g}$$

$0.4 \sim 0.5$  if sharp      "  
 $\sim 0$  if rounded      " regardless

$$h_1 - h_2 = \left( \sum K_L + f \frac{L}{D} \right) \frac{V^2}{2g}$$

$$V = \left\{ 2g(h_1 - h_2) / \left( \sum K_L + f \frac{L}{D} \right) \right\}^{1/2} \quad (1)$$

Assume rounded inlet conditions:  $\sum K_L = K_{L,out} = 1$   
 $h_1 - h_2 = 1$  m. BUT WE DON'T KNOW  $f$ !

$$f = f \left( Re = \frac{VD}{\nu}, \frac{\epsilon/D}{10^{-3}} \right) : \text{MOODY} \quad (2)$$

$\epsilon = 0.26 \text{ mm}, 10^{-3}$

BUT WE DON'T KNOW  $V$  (until we know  $f$ )!

Method 1: Take "standard"  $f = f^{(1)} = 0.02$ ,  
 get  $V = V^{(1)}$  from (1), then  $f = f^{(2)}$  from  
 Moody with  $Re = Re^{(1)} = V^{(1)} D / \nu$ . Go back to  
 (1) etc until  $V^{(n+1)} \sim V^{(n)}$

Method 2: Assume Fully Rough Turbulent Flow  
 and get  $f = f^{(1)} = f(Re \rightarrow \infty, \epsilon/D)$  from Moody,  
 then get  $V = V^{(1)}$  from (1), now  $Re = Re^{(1)} =$   
 $V^{(1)} D / \nu$  & Moody gives  $f = f^{(2)}$  etc.

## Computations for $h_1 - h_2 = 1m$

Cast Iron : Table 8.1  $\epsilon = 0.26 \text{ mm}$

$$\text{Rel. Roughness} = \epsilon/D = 0.26 \text{ mm} / 0.26 \text{ m} = 10^{-3}$$

If flow is R.T.  $\Rightarrow$  Moody gives  $f^{(1)} = 0.0196$

[very close to "standard" value of 0.02 used in Method 1 - here the two approaches are the "same"]

$$V^{(1)} = \frac{\sqrt{2g(h_1 - h_2)}}{\sqrt{1 + f^{(1)}(L/D)}} = \frac{\sqrt{2 \cdot 9.8 \cdot 1}}{\sqrt{1 + 0.0196 \frac{260}{0.26}}} = \frac{\sqrt{19.6}}{\sqrt{1 + 19.6}} = 0.98 \frac{\text{m}}{\text{s}}$$

Now  $Re^{(1)} = V^{(1)} D / \nu = 0.98 \cdot 0.26 / 10^{-6} = 2.5 \cdot 10^5$   
and  $\epsilon/D = 10^{-3}$  gives  
 $f^{(2)} = 0.0209$

$$V^{(2)} = \frac{\sqrt{2g(h_1 - h_2)}}{\sqrt{1 + f^{(2)}(L/D)}} = \frac{\sqrt{19.6}}{\sqrt{1 + 20.9}} = 0.95 \frac{\text{m}}{\text{s}}$$

Now  $Re^{(2)} = V^{(2)} D / \nu = 2.47 \cdot 10^5 \approx 2.5 \cdot 10^5$  same as before. Therefore same  $f^{(3)} = f^{(2)}$ , and we're done!

$$V = V^{(2)} = 0.95 \text{ m/s} \Rightarrow Q = V \cdot A = V \frac{\pi}{4} D^2 = 0.0504 \frac{\text{m}^3}{\text{s}}$$

If  $h_1 - h_2 = 4m$ ! not 1m - quick estimate from above :  $V \propto \sqrt{h_1 - h_2}$  assuming  $f = f^{(2)} \approx 0.02$

$$V_4 = \sqrt{4/1} V_1 = 1.9 \text{ m/s} \Rightarrow Q = 0.1 \text{ m}^3/\text{s}$$

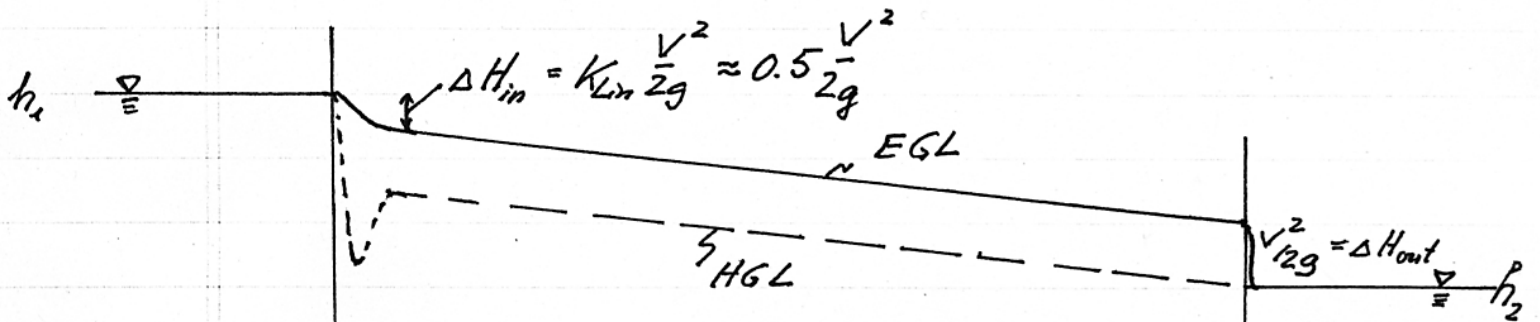
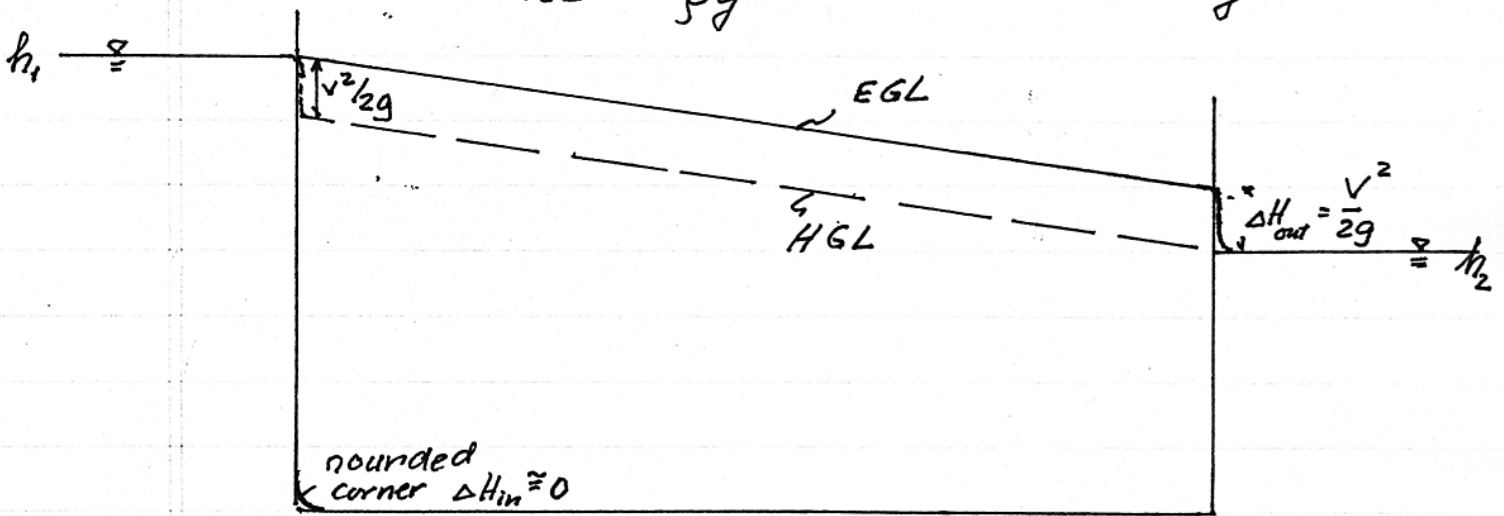
$Re = 4.9 \cdot 10^5$ ;  $\epsilon/D = 10^{-3} \Rightarrow f = 0.0203 \approx 0.0209$  ✓  
 $V_4$  is correct.

EGL: ENERGY GRADE LINE (TOTAL HEAD LINE) &

HGL: HYDRAULIC GRADE LINE (PIEZOMETRIC HEADLINE)

$$\text{EGL @ } z_{\text{EGL}} = H = \frac{P_{CG}}{\rho g} + z_{CG} + \frac{V^2}{2g}$$

$$\text{HGL @ } z_{\text{HGL}} = \frac{P_{CG}}{\rho g} + z_{CG} = H - \frac{V^2}{2g}$$



Note: EGL always going down in direction of flow. Not so for HGL

$$V_1 = V_{vc} \approx 1.6V$$

$$V_1^2/2g = 2.6 V^2/2g$$