

Practice Problems No. 2

(a) xy coordinate system moves with O .

$$\omega|_{\text{cyl.}} = -\dot{\varphi} \hat{e}_z$$

$$\omega|_{\text{block}} = -\dot{\theta} \hat{e}_z$$

No slip between the cylinder and the floor:

$$\underline{v}_O = R\dot{\varphi} \hat{e}_x$$

$$\underline{v}_C = \underline{v}_O + \dot{x}_C \hat{e}_x + \dot{y}_C \hat{e}_y = (R\dot{\varphi} + \dot{x}_C) \hat{e}_x + \dot{y}_C \hat{e}_y$$

No slip between the block and the cylinder:

$$\underline{v}_B|_{\text{cyl.}} = \underline{v}_B|_{\text{block}}$$

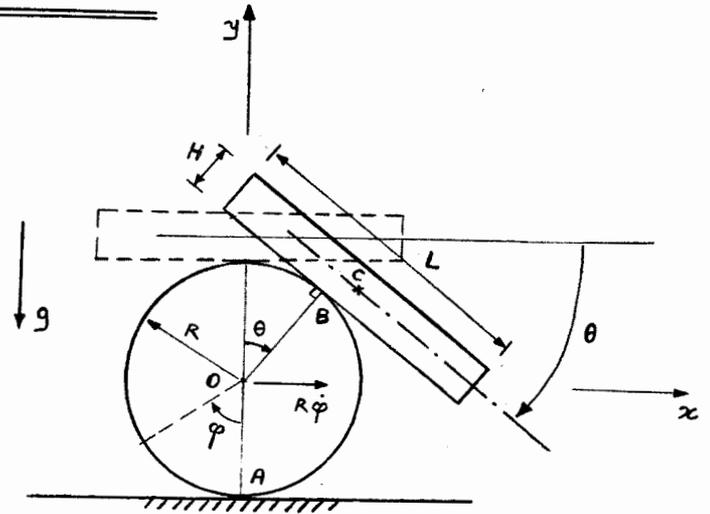
$$\begin{aligned} \underline{v}_B|_{\text{cyl.}} &= \underline{v}_O + \omega|_{\text{cyl.}} \times \underline{OB} = R\dot{\varphi} \hat{e}_x + (-\dot{\varphi} \hat{e}_z) \times (R \sin \theta \hat{e}_x + R \cos \theta \hat{e}_y) \\ &= R\dot{\varphi} (1 + \cos \theta) \hat{e}_x - R\dot{\varphi} \sin \theta \hat{e}_y \end{aligned}$$

$$\underline{v}_B|_{\text{block}} = \underline{v}_C + \omega|_{\text{block}} \times \underline{CB} = (R\dot{\varphi} + \dot{x}_C) \hat{e}_x + \dot{y}_C \hat{e}_y + (-\dot{\theta} \hat{e}_z) \times [(R \sin \theta - x_C) \hat{e}_x + (R \cos \theta - y_C) \hat{e}_y]$$

$$= (R\dot{\varphi} + \dot{x}_C + R\dot{\theta} \cos \theta - \dot{\theta} y_C) \hat{e}_x + (\dot{y}_C - R\dot{\theta} \sin \theta + \dot{\theta} x_C) \hat{e}_y$$

$$\therefore \begin{cases} R\dot{\varphi} \cos \theta = \dot{x}_C + R\dot{\theta} \cos \theta - \dot{\theta} y_C \\ -R\dot{\varphi} \sin \theta = \dot{y}_C - R\dot{\theta} \sin \theta + \dot{\theta} x_C \end{cases} \Rightarrow \begin{cases} \sin \theta \dot{x}_C + \cos \theta \dot{y}_C - \dot{\theta} y_C \sin \theta + \dot{\theta} x_C \cos \theta = 0 \\ R(\dot{\varphi} - \dot{\theta}) = \cos \theta \dot{x}_C - \sin \theta \dot{y}_C - \dot{\theta} \cos \theta y_C - \dot{\theta} x_C \sin \theta \end{cases}$$

$$\Rightarrow \begin{cases} \frac{d}{dt} (\sin \theta x_C + \cos \theta y_C) = 0 \\ \frac{d}{dt} [R(\varphi - \theta)] = \frac{d}{dt} [\cos \theta x_C - \sin \theta y_C] \end{cases}$$



Problem 1

(a) $\Rightarrow \begin{cases} \sin\theta x_c + \cos\theta y_c = \text{Const.} = R + \frac{H}{2} \\ \cos\theta x_c - \sin\theta y_c = R(\varphi - \theta) \end{cases} \quad \left(\text{at } \theta=0, x_c=0, y_c=R + \frac{H}{2} \right)$

$\rightarrow \begin{cases} x_c = R(\varphi - \theta) \cos\theta + (R + \frac{H}{2}) \sin\theta \\ y_c = (R + \frac{H}{2}) \cos\theta - R(\varphi - \theta) \sin\theta \end{cases}$

(b)

$f_1 = \varphi$ & $f_2 = \theta$ complete set of independent generalized coordinates

Mass of the block = m

$I_{\text{block}} = \frac{1}{12} m (H^2 + L^2)$

$T = \frac{1}{2} M \dot{v}_0 \cdot \dot{v}_0 + \frac{1}{2} I_{\text{cyl}} \omega_{\text{cyl}} \cdot \omega_{\text{cyl}} + \frac{1}{2} m \dot{v}_c \cdot \dot{v}_c + \frac{1}{2} I_{\text{block}} \omega_{\text{block}} \cdot \omega_{\text{block}}$

$T = \frac{1}{2} M (R\dot{\varphi})^2 + \frac{1}{2} (\frac{1}{2} MR^2) \dot{\varphi}^2 + \frac{1}{2} m [(R\dot{\varphi} + \dot{x}_c)^2 + \dot{y}_c^2] + \frac{1}{2} [\frac{1}{12} m (H^2 + L^2)] \dot{\theta}^2$

$V = mg y_c + Mg y_0^0 = mg y_c$

$T = \frac{3}{4} MR^2 \dot{\varphi}^2 + \frac{1}{2} m [R^2 \dot{\varphi}^2 + R^2 (\varphi - \theta)^2 \dot{\theta}^2 + R^2 (\dot{\varphi} - \dot{\theta})^2 + (R + \frac{H}{2})^2 \dot{\theta}^2 - 2R^2 \dot{\theta} \dot{\varphi} (\varphi - \theta) \sin\theta + 2R^2 \dot{\varphi}^2 \cos\theta + RH \dot{\varphi} \dot{\theta} \cos\theta + 2R(\dot{\varphi} - \dot{\theta})(R + \frac{H}{2}) \dot{\theta}] + \frac{1}{24} m (H^2 + L^2) \dot{\theta}^2$

$V = mg \left[(R + \frac{H}{2}) \cos\theta - R(\varphi - \theta) \sin\theta \right]$

$\mathcal{L} = T - V$

$\frac{\partial \mathcal{L}}{\partial \dot{\varphi}} = \frac{3}{2} MR^2 \dot{\varphi} + \frac{m}{2} [2R^2 \dot{\varphi} + 2R^2 (\dot{\varphi} - \dot{\theta}) - 2R^2 (\varphi - \theta) \dot{\theta} \sin\theta + 2R^2 (2\dot{\varphi}) \cos\theta + RH \dot{\theta} \cos\theta + 2R(R + \frac{H}{2}) \dot{\theta}]$

$\frac{\partial \mathcal{L}}{\partial \varphi} = \frac{m}{2} [2R^2 \dot{\theta}^2 (\varphi - \theta) - 2R^2 \dot{\theta} \dot{\varphi} \sin\theta] + mgR \sin\theta$

Problem 1

(b)

$$\frac{\partial \mathcal{L}}{\partial \dot{\theta}} = \frac{m}{2} \left[2R^2(\varphi - \theta)^2 \dot{\theta} - 2R^2(\dot{\varphi} - \dot{\theta}) + 2\left(R + \frac{H}{2}\right)^2 \dot{\theta} - 2R^2 \dot{\varphi} (\varphi - \theta) \sin \theta + RH \dot{\varphi} \cos \theta + 2R(\dot{\varphi} - 2\dot{\theta})\left(R + \frac{H}{2}\right) \right] + \frac{m}{12}(H^2 + L^2)\dot{\theta}$$

$$\frac{\partial \mathcal{L}}{\partial \theta} = \frac{m}{2} \left[2R^2 \dot{\theta}^2 (\theta - \varphi) - 2R^2 \dot{\theta} \dot{\varphi} \varphi \cos \theta + 2R^2 \dot{\theta} \dot{\varphi} \sin \theta + 2R^2 \dot{\theta} \dot{\varphi} \theta \cos \theta - 2R^2 \dot{\varphi}^2 \sin \theta - RH \dot{\varphi} \dot{\theta} \sin \theta \right] + mg\left(R + \frac{H}{2}\right) \sin \theta + mgR \varphi \cos \theta - mgR \theta \cos \theta - mgR \sin \theta$$

$$\underline{\underline{\varphi}} = \underline{\underline{\theta}} = 0$$

$$\underline{\underline{\delta \varphi}}: \left[\frac{3}{2} MR^2 + 2mR^2(1 + \cos \theta) \right] \ddot{\varphi} + \left[-mR^2(\varphi - \theta) \sin \theta + \frac{mRH}{2}(\cos \theta + 1) \right] \ddot{\theta} + \dots - mgR \sin \theta = 0$$

$$\underline{\underline{\delta \theta}}: \left[mR^2(\varphi - \theta)^2 + m\left(\frac{H^2}{3} + \frac{L^2}{12}\right) \right] \ddot{\theta} + \left[-mR^2(\varphi - \theta) \sin \theta + \frac{mRH}{2}(\cos \theta + 1) \right] \ddot{\varphi} + \dots - mg\frac{H}{2} \sin \theta + mgR \cos \theta (\theta - \varphi) = 0$$

Governing Equations of Motion

Note that missing terms do not play a role in the following linear stability analysis.

(c)

θ and φ are non-ignorable coordinates. \Rightarrow equilibrium position $\theta = \theta_s$
 $\varphi = \varphi_s$

$$\delta \varphi \Rightarrow -mgR \sin \theta_s = 0 \quad \rightarrow \quad \theta_s = 0 \quad \theta_s = \pi$$

$$\delta \theta \Rightarrow -mg\frac{H}{2} \sin \theta_s + mgR \cos \theta_s (\theta_s - \varphi_s) = 0 \quad \xrightarrow{\theta_s = 0} \quad \varphi_s = 0$$

\therefore Equilibrium position $\underline{\underline{\theta_s = \varphi_s = 0}}$

Problem 1

(c)

$$\theta = \theta_s + \varepsilon_\theta(t) = \varepsilon_\theta(t)$$

$$\varphi = \varphi_s + \varepsilon_\varphi(t) = \varepsilon_\varphi(t)$$

$$\delta\theta \Rightarrow (4mR^2 + \frac{3}{2}MR^2) \ddot{\varepsilon}_\varphi + mRH \ddot{\varepsilon}_\theta - mgR \varepsilon_\theta = 0$$

$$\delta\varphi \Rightarrow (m\frac{H^2}{3} + m\frac{L^2}{12}) \ddot{\varepsilon}_\theta + mRH \ddot{\varepsilon}_\varphi + mg(R - \frac{H}{2}) \varepsilon_\theta - mgR \varepsilon_\varphi = 0$$

$$\begin{bmatrix} R^2(\frac{3}{2}M + 4m) & mRH \\ mRH & m(\frac{H^2}{3} + \frac{L^2}{12}) \end{bmatrix} \begin{Bmatrix} \ddot{\varepsilon}_\varphi \\ \ddot{\varepsilon}_\theta \end{Bmatrix} + \begin{bmatrix} 0 & -mgR \\ -mgR & mg(R - \frac{H}{2}) \end{bmatrix} \begin{Bmatrix} \varepsilon_\varphi \\ \varepsilon_\theta \end{Bmatrix} = \begin{Bmatrix} 0 \\ 0 \end{Bmatrix}$$

$$\text{Let } \begin{Bmatrix} \varepsilon_\varphi \\ \varepsilon_\theta \end{Bmatrix} = \begin{Bmatrix} x \\ y \end{Bmatrix} e^{i\omega t}$$

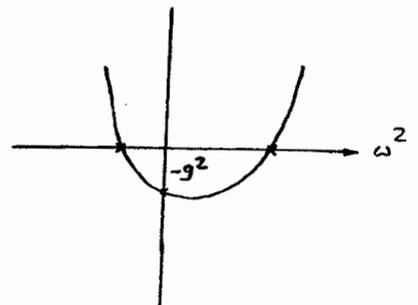
$$\therefore \begin{bmatrix} -\omega^2 R^2(\frac{3}{2}M + 4m) & -mR(g + \omega^2 H) \\ -mR(g + \omega^2 H) & +mg(R - \frac{H}{2}) - \omega^2 m(\frac{H^2}{3} + \frac{L^2}{12}) \end{bmatrix} \begin{Bmatrix} x \\ y \end{Bmatrix} = \begin{Bmatrix} 0 \\ 0 \end{Bmatrix}$$

$$\text{Non-trivial solution: } \begin{vmatrix} -\omega^2 R^2(\frac{3}{2}M + 4m) & -mR(g + \omega^2 H) \\ -mR(g + \omega^2 H) & mg(R - \frac{H}{2}) - \omega^2 m(\frac{H^2}{3} + \frac{L^2}{12}) \end{vmatrix} = 0$$

$$\Rightarrow \left[\frac{M}{m} \left(\frac{H^2}{2} + \frac{L^2}{8} \right) + \frac{1}{3} (H^2 + L^2) \right] \omega^4 + \left[-4Rg + \frac{3}{4} \frac{M}{m} Hg - \frac{3}{2} \frac{M}{m} Rg \right] \omega^2 - g^2 = 0$$

This equation has a negative root for ω^2 .

So $\varphi_s = \theta_s = 0$ is unstable.



Problem 1

Note that in the limiting case of $\frac{M}{m} \rightarrow \infty$, the problem turns into the rolling block on a fixed cylinder. In that case, we get

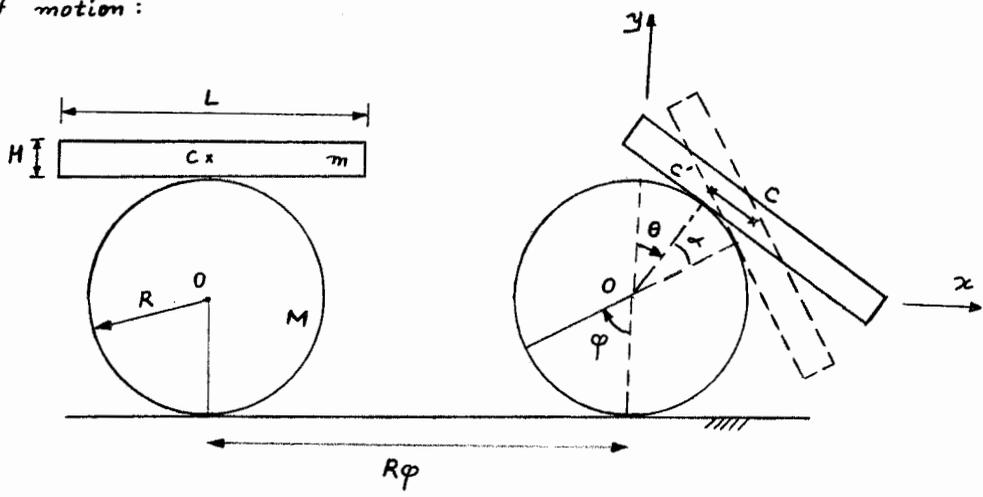
$$\left[\left(\frac{H^2}{2} + \frac{L^2}{8} \right) + \frac{1}{3} \frac{m}{M} (H^2 + L^2) \right] \omega^4 + \left[-4Rg \frac{m}{M} + \frac{3}{4} Hg - \frac{3}{2} Rg \right] \omega^2 - \frac{m}{M} g^2 = 0$$

$$\left(\frac{H^2}{2} + \frac{L^2}{8} \right) \omega^2 + \frac{3}{4} g (H - 2R) = 0$$

$$\rightarrow \omega^2 = \frac{\frac{3}{4} g (2R - H)}{\frac{H^2}{2} + \frac{L^2}{8}} \rightarrow \begin{cases} \theta_s = 0 \text{ is stable for } \underline{H < 2R} \\ \theta_s = 0 \text{ is unstable for } \underline{H > 2R} \end{cases}$$

* Another way to find the location of the center of mass C of the block (x_c, y_c) by studying the geometry of motion:

To analyze the block rolling constraint, assume the block is glued to the cylinder during rotation φ and then rolls back to location θ .



Let roll-back angle be α ($\alpha = \varphi - \theta$). The absolute rotation of the block is through angle θ .

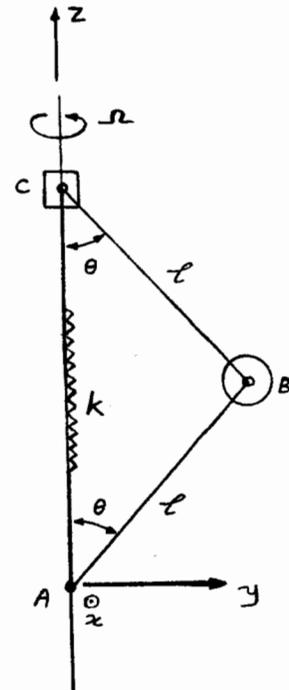
Location of center of mass of the block: (No slip between the block and the cylinder yields

$$CC' = R\alpha = R(\varphi - \theta) \quad \begin{cases} x_c = \left(R + \frac{H}{2} \right) \sin \theta + R(\varphi - \theta) \cos \theta \\ y_c = \left(R + \frac{H}{2} \right) \cos \theta - R(\varphi - \theta) \sin \theta \end{cases}$$

Problem 2

xyz rotates about Z with Ω .

Assume small disk B and slider C are point masses.



$$\underline{r}_{AC} = 2l \cos\theta \hat{e}_z$$

$$\underline{v}_C = \frac{d}{dt} \underline{r}_{AC} = -2l \sin\theta \dot{\theta} \hat{e}_z$$

$$\underline{v}_B = \frac{d}{dt} \underline{r}_{AB} = \underline{\omega}_{AB} \times \underline{r}_{AB}$$

$$\underline{\omega}_{AB} = -\dot{\theta} \hat{e}_x + \Omega \hat{e}_z$$

$$\underline{r}_{AB} = l \sin\theta \hat{e}_y + l \cos\theta \hat{e}_z$$

$$\underline{v}_B = (-\dot{\theta} \hat{e}_x + \Omega \hat{e}_z) \times (l \sin\theta \hat{e}_y + l \cos\theta \hat{e}_z) = -\dot{\theta} l \sin\theta \hat{e}_z + \dot{\theta} l \cos\theta \hat{e}_y - \Omega l \sin\theta \hat{e}_x$$

$$T = \frac{1}{2} m \underline{v}_B \cdot \underline{v}_B + \frac{1}{2} m \underline{v}_C \cdot \underline{v}_C = \frac{1}{2} m (\dot{\theta}^2 l^2 + \Omega^2 l^2 \sin^2\theta) + \frac{1}{2} m (4l^2 \sin^2\theta \dot{\theta}^2)$$

$$V = mgy_C + mgy_B + \frac{1}{2} k(2l - 2l \cos\theta)^2 = 3mgl \cos\theta + 2kl^2(1 - \cos\theta)^2$$

$$\mathcal{L} = T - V$$

$$\delta\theta: \quad \frac{d}{dt} (m l^2 \dot{\theta} + 4m l^2 \dot{\theta} \sin^2\theta) - m \Omega^2 l^2 \sin\theta \cos\theta - 4m l^2 \dot{\theta}^2 \sin\theta \cos\theta - 3mgl \sin\theta + 4kl^2(1 - \cos\theta) \sin\theta = 0$$

θ is non-ignorable \Rightarrow Steady motion $\theta = \theta_s$

$$\left[-m\Omega^2 l^2 \cos\theta_s - 3mg l + 4kl^2 (1 - \cos\theta_s) \right] \sin\theta_s = 0$$

(A) $\sin\theta_s = 0 \rightarrow \theta_s = 0 \quad \left(0 \leq \theta_s < \frac{\pi}{2} \right)$

(B) $\cos\theta_s = \frac{4kl^2 - 3mg l}{4kl^2 + m\Omega^2 l^2} \rightarrow \theta_s = \cos^{-1} \left(\frac{4kl^2 - 3mg l}{4kl^2 + m\Omega^2 l^2} \right)$

possible only if $0 < \frac{4kl^2 - 3mg l}{4kl^2 + m\Omega^2 l^2} < 1 \Rightarrow k > \frac{3mg}{4l}$

Stability of θ_s :

Hold Ω constant and supply τ_z as needed.

$$\theta = \theta_s + \varepsilon(t)$$

$$\begin{aligned} \delta\theta \Rightarrow & m l^2 \ddot{\varepsilon} + 4m l^2 \sin^2\theta_s \ddot{\varepsilon} - \cancel{m\Omega^2 l^2 \sin\theta_s \cos\theta_s} - m\Omega^2 l^2 \cos^2\theta_s \varepsilon + m\Omega^2 l^2 \sin^2\theta_s \varepsilon \\ & - \cancel{3mg l \sin\theta_s} - 3mg l \cos\theta_s \varepsilon + 4kl^2 (1 - \cos\theta_s) \sin\theta_s + 4kl^2 \sin^2\theta_s \varepsilon \\ & + 4kl^2 \cos\theta_s (1 - \cos\theta_s) \varepsilon = 0 \end{aligned}$$

O(1) terms cancel. $\left\{ \left[-m\Omega^2 l^2 \cos\theta_s - 3mg l + 4kl^2 (1 - \cos\theta_s) \right] \sin\theta_s = 0 \right\}$

$$\rightarrow m l^2 (1 + 4\sin^2\theta_s) \ddot{\varepsilon} + \left[m\Omega^2 l^2 (\sin^2\theta_s - \cos^2\theta_s) - 3mg l \cos\theta_s + 4kl^2 (\sin^2\theta_s - \cos^2\theta_s) + 4l^2 \cos\theta_s \right] \varepsilon = 0$$

(A) $m l^2 \ddot{\varepsilon} + (-m\Omega^2 l^2 - 3mg l) \varepsilon = 0 \rightarrow \theta_s = 0$ is unstable.

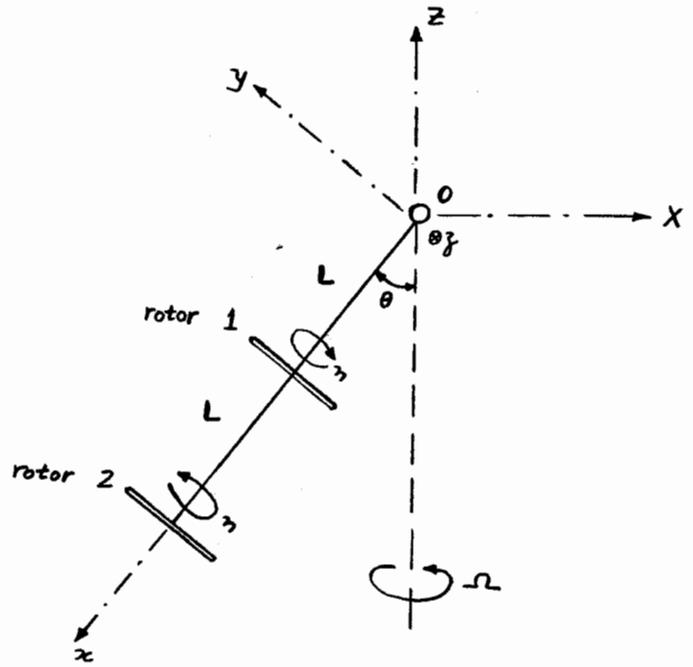
(B) $m l^2 (1 + 4\sin^2\theta_s) \ddot{\varepsilon} + (m\Omega^2 l^2 \sin^2\theta_s + 4kl^2 \sin^2\theta_s) \varepsilon = 0$

$$\rightarrow \ddot{\varepsilon} + \left(\frac{\Omega^2 + 4k/m}{1 + 4\sin^2\theta_s} \right) \sin^2\theta_s \varepsilon = 0 \rightarrow \theta_s = \cos^{-1} \left(\frac{4kl^2 - 3mg l}{4kl^2 + m\Omega^2 l^2} \right) \text{ is stable.}$$

exists only if $k > \frac{3mg}{4l}$

Problem 3

xyz coordinate system rotates with Ω about z.



$$\tau_0 = \frac{d}{dt} H_0 \quad (\dot{\alpha}_0 = 0)$$

$$H_0 = [I]_0 \omega$$

$$H_0 = [I]_{0|1} \omega|_1 + [I]_{0|2} \omega|_2$$

$$[I]_{0|1} = \begin{bmatrix} I_3 & 0 & 0 \\ 0 & I_1 & 0 \\ 0 & 0 & I_1 \end{bmatrix} + m \begin{bmatrix} 0 & 0 & 0 \\ 0 & L^2 & 0 \\ 0 & 0 & L^2 \end{bmatrix} = \begin{bmatrix} I_3 & 0 & 0 \\ 0 & I_1 + mL^2 & 0 \\ 0 & 0 & I_1 + mL^2 \end{bmatrix}$$

$$[I]_{0|2} = \begin{bmatrix} I_3 & 0 & 0 \\ 0 & I_1 + 4mL^2 & 0 \\ 0 & 0 & I_1 + 4mL^2 \end{bmatrix}$$

$$\omega|_1 = \dot{\psi}_1 \hat{e}_x + \Omega \hat{e}_z = \dot{\psi}_1 \hat{e}_x + \Omega (-\cos\theta \hat{e}_x + \sin\theta \hat{e}_y) = \underbrace{(\dot{\psi}_1 - \Omega \cos\theta)}_n \hat{e}_x + \Omega \sin\theta \hat{e}_y$$

$$\omega|_2 = \dot{\psi}_2 \hat{e}_x + \Omega \hat{e}_z = \dot{\psi}_2 \hat{e}_x + \Omega (-\cos\theta \hat{e}_x + \sin\theta \hat{e}_y) = \underbrace{(\dot{\psi}_2 - \Omega \cos\theta)}_{-n} \hat{e}_x + \Omega \sin\theta \hat{e}_y$$

(Note that, in this problem, n is absolute spin about x)

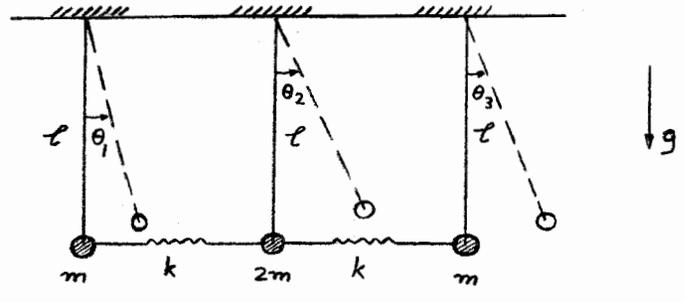
$$\therefore H_0 = \begin{bmatrix} I_3 & 0 & 0 \\ 0 & I_1 + mL^2 & 0 \\ 0 & 0 & I_1 + mL^2 \end{bmatrix} \begin{Bmatrix} n \\ -\Omega \sin\theta \\ 0 \end{Bmatrix} + \begin{bmatrix} I_3 & 0 & 0 \\ 0 & I_1 + 4mL^2 & 0 \\ 0 & 0 & I_1 + 4mL^2 \end{bmatrix} \begin{Bmatrix} -n \\ -\Omega \sin\theta \\ 0 \end{Bmatrix} = (2I_1 + 5mL^2) \Omega \sin\theta \hat{e}_y$$

$$\tau_0 = \frac{d}{dt} H_0 = \Omega \sin\theta (2I_1 + 5mL^2) \frac{d}{dt} \hat{e}_y = \underbrace{-\Omega^2 \sin\theta \cos\theta (2I_1 + 5mL^2)}_{\substack{\text{required torque} \\ \text{to maintain} \\ \text{the precession}}} \hat{e}_z$$

$$\Omega \hat{e}_z \times \hat{e}_y = -\Omega \cos\theta \hat{e}_z$$

Problem 4

$$T = \frac{1}{2} m (\ell \dot{\theta}_1)^2 + \frac{1}{2} (2m) (\ell \dot{\theta}_2)^2 + \frac{1}{2} m (\ell \dot{\theta}_3)^2$$



For θ_1, θ_2 and θ_3 small, potential energy correct to quadratic terms in $\theta_1, \theta_2, \theta_3$:

$$V \approx mg \frac{\ell}{2} \theta_1^2 + 2mg \frac{\ell}{2} \theta_2^2 + mg \frac{\ell}{2} \theta_3^2 + \frac{1}{2} k \ell^2 (\theta_2 - \theta_1)^2 + \frac{1}{2} k \ell^2 (\theta_3 - \theta_2)^2$$

$$\bar{\theta}_1 = \bar{\theta}_2 = \bar{\theta}_3 = 0$$

$$\mathcal{L} = T - V$$

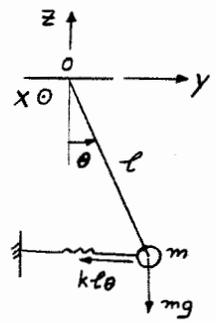
$$\begin{matrix} \delta\theta_1: \\ \delta\theta_2: \\ \delta\theta_3: \end{matrix} \ell^2 \begin{bmatrix} m & 0 & 0 \\ 0 & 2m & 0 \\ 0 & 0 & m \end{bmatrix} \begin{Bmatrix} \ddot{\theta}_1 \\ \ddot{\theta}_2 \\ \ddot{\theta}_3 \end{Bmatrix} + \ell^2 \begin{bmatrix} \frac{mg}{\ell} + k & -k & 0 \\ -k & 2\frac{mg}{\ell} + 2k & -k \\ 0 & -k & \frac{mg}{\ell} + k \end{bmatrix} \begin{Bmatrix} \theta_1 \\ \theta_2 \\ \theta_3 \end{Bmatrix} = \begin{Bmatrix} 0 \\ 0 \\ 0 \end{Bmatrix}$$

$$\rightarrow [M] = \ell^2 \begin{bmatrix} m & 0 & 0 \\ 0 & 2m & 0 \\ 0 & 0 & m \end{bmatrix}, \quad [K] = \ell^2 \begin{bmatrix} \frac{mg}{\ell} + k & -k & 0 \\ -k & 2\frac{mg}{\ell} + 2k & -k \\ 0 & -k & \frac{mg}{\ell} + k \end{bmatrix}$$

For a single pendulum (θ small):

$$\underline{\dot{z}}_0 = \frac{d}{dt} \underline{H}_0 \quad \underline{H}_0 = [I]_0 \underline{\omega} = m \ell^2 \dot{\theta} \hat{e}_x$$

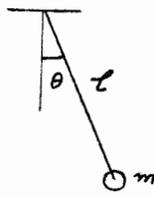
$$\underline{\dot{z}}_0 = - (mg \ell \theta + k \ell^2 \theta) \hat{e}_x$$



Problem 4

$$\Rightarrow \ddot{\theta} + \left(\frac{g}{\ell} + \frac{k}{m}\right)\theta = 0 \quad \rightarrow \quad \omega^2 = \frac{g}{\ell} + \frac{k}{m}$$

and if there is no spring: $\omega^2 = \frac{g}{\ell}$



So it is easy to guess two of the modes:

$$\{a\}_1 = \begin{Bmatrix} 1 \\ 1 \\ 1 \end{Bmatrix} \quad \omega_1^2 = \frac{g}{\ell}$$

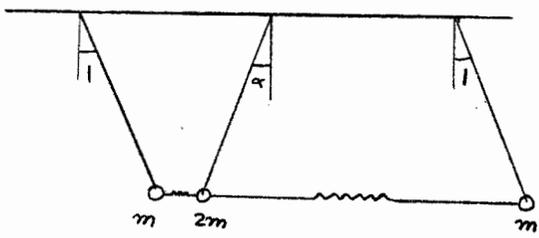
In this mode, springs do not come into play and pendulums oscillate with frequency $\omega = \sqrt{\frac{g}{\ell}}$.

$$\{a\}_2 = \begin{Bmatrix} 1 \\ 0 \\ -1 \end{Bmatrix} \quad \omega_2^2 = \frac{g}{\ell} + \frac{k}{m}$$

In this mode, mass 2m is stationary and outer masses always move in opposite directions.

To find the third mode:

Try $\{a\}_3 = \begin{Bmatrix} 1 \\ -\alpha \\ 1 \end{Bmatrix}$



Using orthogonality of mode shapes:

$$\{a\}_3^T [M] \{a\}_1 = 0$$

$$\rightarrow \{1 \quad -\alpha \quad 1\} \begin{bmatrix} 1 & 0 & 0 \\ 0 & 2 & 0 \\ 0 & 0 & 1 \end{bmatrix} (m\ell^2) \begin{Bmatrix} 1 \\ 1 \\ 1 \end{Bmatrix} = 0 \quad \rightarrow \quad \alpha = 1$$

Problem 4

$$\therefore \{a\}_3 = \begin{Bmatrix} 1 \\ -1 \\ 1 \end{Bmatrix}$$

$$\omega_3^2 = \frac{\{a\}_3^t [K] \{a\}_3}{\{a\}_3^t [M] \{a\}_3} = \frac{\{1 \ -1 \ 1\} \begin{bmatrix} \frac{mg}{\ell} + k & -k & 0 \\ -k & \frac{2mg}{\ell} + 2k & -k \\ 0 & -k & \frac{mg}{\ell} + k \end{bmatrix} (\ell^2) \begin{Bmatrix} 1 \\ -1 \\ 1 \end{Bmatrix}}{\{1 \ -1 \ 1\} \begin{bmatrix} 1 & 0 & 0 \\ 0 & 2 & 0 \\ 0 & 0 & 1 \end{bmatrix} (m\ell^2) \begin{Bmatrix} 1 \\ -1 \\ 1 \end{Bmatrix}}$$

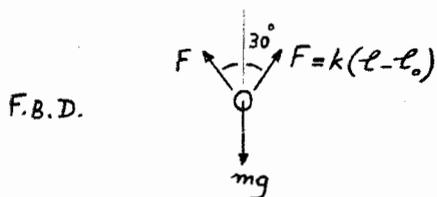
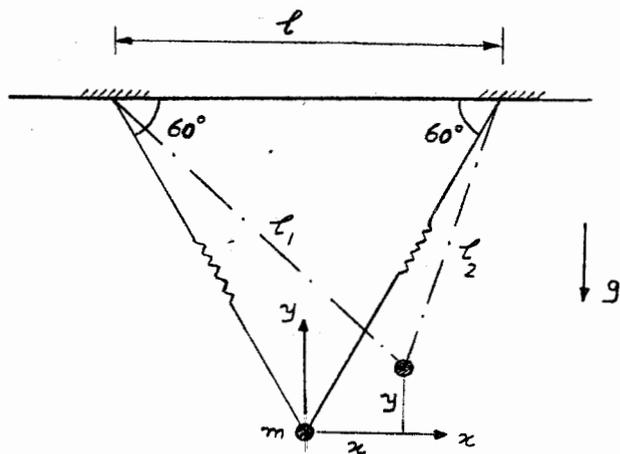
$$\omega_3^2 = \frac{g}{\ell} + 2 \frac{k}{m}$$

Problem 5

Free length of the springs: l_0

Length of the springs at static equilibrium: l

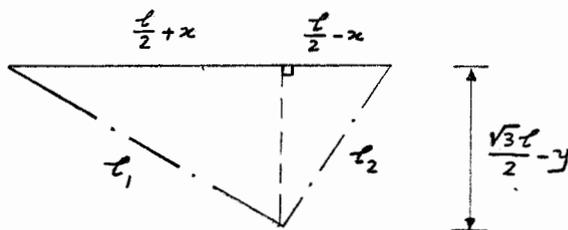
At static equilibrium:



$$mg = 2F \cos 30^\circ = \sqrt{3}F = \sqrt{3}k(l - l_0)$$

$$l_1 = \sqrt{\left(\frac{l}{2} + x\right)^2 + \left(\frac{\sqrt{3}l}{2} - y\right)^2}$$

$$l_2 = \sqrt{\left(\frac{l}{2} - x\right)^2 + \left(\frac{\sqrt{3}l}{2} - y\right)^2}$$



$$T = \frac{1}{2}m(\dot{x}^2 + \dot{y}^2)$$

$$V = mgy + \frac{1}{2}k(l_1 - l_0)^2 + \frac{1}{2}k(l_2 - l_0)^2$$

$$V = mgy + \frac{1}{2}k[l_1^2 + l_2^2 - 2l_0(l_1 + l_2) + 2l_0^2]$$

$$V = mgy + k[l^2 + x^2 + y^2 - \sqrt{3}ly - l_0(l_1 + l_2) + l_0^2]$$

$$\mathcal{L} = T - V \quad \bar{x} = \bar{y}$$

$$\underline{\underline{\delta x}}: \quad m\ddot{x} + k \left[2x - \ell_0 \left(\frac{\partial \ell_1}{\partial x} + \frac{\partial \ell_2}{\partial x} \right) \right] = 0$$

$$\underline{\underline{\delta y}}: \quad m\ddot{y} + k \left[2y - \sqrt{3}\ell - \ell_0 \left(\frac{\partial \ell_1}{\partial y} + \frac{\partial \ell_2}{\partial y} \right) \right] + mg = 0$$

$$\frac{\partial \ell_1}{\partial x} = \left(\frac{\ell}{2} + x \right) \left[\left(\frac{\ell}{2} + x \right)^2 + \left(\frac{\sqrt{3}}{2}\ell - y \right)^2 \right]^{-\frac{1}{2}}$$

$$\frac{\partial \ell_2}{\partial x} = - \left(\frac{\ell}{2} - x \right) \left[\left(\frac{\ell}{2} - x \right)^2 + \left(\frac{\sqrt{3}}{2}\ell - y \right)^2 \right]^{-\frac{1}{2}}$$

$$\frac{\partial \ell_1}{\partial y} = - \left(\frac{\sqrt{3}}{2}\ell - y \right) \left[\left(\frac{\ell}{2} + x \right)^2 + \left(\frac{\sqrt{3}}{2}\ell - y \right)^2 \right]^{-\frac{1}{2}}$$

$$\frac{\partial \ell_2}{\partial y} = - \left(\frac{\sqrt{3}}{2}\ell - y \right) \left[\left(\frac{\ell}{2} - x \right)^2 + \left(\frac{\sqrt{3}}{2}\ell - y \right)^2 \right]^{-\frac{1}{2}}$$

x, y are small and one can approximate the derivatives using Taylor series expansion:

$$f(a, b) = (a^2 + b^2)^{-\frac{1}{2}}$$

$$\begin{aligned} f(a_0 + \Delta a, b_0 + \Delta b) &= \left[(a_0 + \Delta a)^2 + (b_0 + \Delta b)^2 \right]^{-\frac{1}{2}} = f(a_0, b_0) + \frac{\partial f}{\partial a} \Big|_{\substack{a=a_0 \\ b=b_0}} \Delta a + \frac{\partial f}{\partial b} \Big|_{\substack{a=a_0 \\ b=b_0}} \Delta b \\ &\approx (a_0^2 + b_0^2)^{-\frac{1}{2}} - a_0 (a_0^2 + b_0^2)^{-\frac{3}{2}} \Delta a - b_0 (a_0^2 + b_0^2)^{-\frac{3}{2}} \Delta b \end{aligned}$$

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$$\begin{aligned} \frac{\partial \ell_1}{\partial x} &= \left(\frac{\ell}{2} + x \right) \left[\ell^{-1} - \frac{\ell^{-2}}{2} x + \frac{\sqrt{3}}{2} \ell^{-2} y + \dots \right] \approx \frac{1}{2} - \frac{1}{4} \frac{x}{\ell} + \frac{\sqrt{3}}{4} \frac{y}{\ell} + \frac{x}{\ell} \\ &\quad \left(a_0 = \frac{\ell}{2}, \quad b_0 = \frac{\sqrt{3}}{2}\ell, \quad \Delta a = x, \quad \Delta b = -y \right) \end{aligned}$$

$$\begin{aligned} \frac{\partial \ell_2}{\partial x} &\approx - \left(\frac{1}{2} + \frac{1}{4} \frac{x}{\ell} + \frac{\sqrt{3}}{4} \frac{y}{\ell} - \frac{x}{\ell} \right) & \left(a_0 = \frac{\ell}{2}, \quad b_0 = \frac{\sqrt{3}}{2}\ell, \quad \Delta a = -x, \quad \Delta b = -y \right) \end{aligned}$$

$$\frac{\partial \ell_1}{\partial y} = - \left(\frac{\sqrt{3}}{2}\ell - y \right) \left[\ell^{-1} - \frac{\ell^{-2}}{2} x + \frac{\sqrt{3}}{2} \ell^{-2} y + \dots \right] \approx - \frac{\sqrt{3}}{2} + \frac{\sqrt{3}}{4} \frac{x}{\ell} - \frac{3}{4} \frac{y}{\ell} + \frac{y}{\ell}$$

$$\frac{\partial \ell_2}{\partial y} \approx -\frac{\sqrt{3}}{2} - \frac{\sqrt{3}}{4} \frac{x}{\ell} - \frac{3}{4} \frac{y}{\ell} + \frac{y}{\ell}$$

∴

$$\begin{aligned} \delta x: & \quad \begin{bmatrix} m & 0 \\ 0 & m \end{bmatrix} \begin{Bmatrix} \ddot{x} \\ \ddot{y} \end{Bmatrix} + k \begin{bmatrix} 2 - \frac{3}{2} \frac{\ell_0}{\ell} & 0 \\ 0 & 2 - \frac{1}{2} \frac{\ell_0}{\ell} \end{bmatrix} \begin{Bmatrix} x \\ y \end{Bmatrix} = \begin{Bmatrix} 0 \\ 0 \end{Bmatrix} \\ \delta y: & \end{aligned}$$

(Note that $mg - \sqrt{3}k(\ell - \ell_0) = 0$ in δy)

$$\Rightarrow \quad [M] = \begin{bmatrix} m & 0 \\ 0 & m \end{bmatrix}, \quad [K] = \begin{bmatrix} 2 - \frac{3}{2} \frac{\ell_0}{\ell} & 0 \\ 0 & 2 - \frac{1}{2} \frac{\ell_0}{\ell} \end{bmatrix} k$$

$$-\omega^2 [M] + [K] = 0 \quad \Rightarrow \quad \begin{cases} \omega_1^2 = \left(2 - \frac{3}{2} \frac{\ell_0}{\ell}\right) \frac{k}{m}, & \{a\}_1 = \begin{Bmatrix} 1 \\ 0 \end{Bmatrix} \\ \omega_2^2 = \left(2 - \frac{1}{2} \frac{\ell_0}{\ell}\right) \frac{k}{m}, & \{a\}_2 = \begin{Bmatrix} 0 \\ 1 \end{Bmatrix} \end{cases}$$